

BRAKE DISK COMPRISING A FRICTION RING AND A BRAKE DISK POT

BACKGROUND OF THE INVENTION

Field of the invention

[0002] The invention relates to a brake disk according to the pre-characterizing portion of claim 1.

Related Art of the Invention

[0003] Recently tribologically highly resistant materials are increasingly used for modern high performance brake disks. Here for example metal-matrix-composites (MMC) or carbon fiber reinforced ceramics on the bases of silicon carbide can be utilized. For car racing, friction rings based on carbon fiber reinforced carbon (C/C) are preferably utilized. Suchlike materials necessitate the usage of different materials for the friction ring and the brake disk hub. Friction ring and brake disk hub together constitute the brake disk.

[0004] Various proposals have already been disclosed for the connection between friction ring and brake disk hub. A commonly utilized connection between the friction ring and the brake disk hub is a conventional screw connection. An example for such a screw connection is disclosed in DE 94 22 141 U1. Another form of connection between friction ring and brake disk hub is the riveted joint, as disclosed in EP 872 659 A1.

[0005] Both aforementioned constructions have in common, that the friction ring as well as the brake disk hub feature ring lands, which are layered above each other concentrically, and are connected by a joining arrangement. Usually bores are featured in the ring lands through which the joining arrangement passes. The disadvantage of this construction is that during the transmission

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of the braking torque flexural stress is induced into the ring land of the friction ring. Such a flexural stress may cause fissures in the comparatively brittle ceramic friction ring. Therefore a huge design-engineering effort is necessary to prevent fissuring in the friction ring.

SUMMARY OF THE INVENTION

[0006] The objective of the invention is to create a brake disk with a separate friction ring and brake disk hub, in which the stresses which act upon the friction ring through the joining arrangement are reduced in comparison to the state of the art.

[0007] According to the invention this objective is accomplished by a brake disk according to the characterizing portion of claim 1.

[0008] The brake disk according to the invention features a friction ring and a brake disk hub. Here the term brake disk hub is understood as a linking element in general, which constitutes the connection between the friction ring and the hub of the wheel. In the case that the friction ring is directly attached to the hub of the wheel, the wheel hub itself is regarded as the brake disk hub according to the invention. The friction ring is understood as the part of the brake disk which contacts the brake pads in a retarding engagement.

[0009] The brake disk hub as well as the friction ring features a concentric ring land. Both ring lands feature a number of recesses. One joining arrangement passes through each of the recesses of the ring lands of the friction ring and the brake disk hub.

[00010] The invention is characterized by a support ring which is arranged such that it encloses the ring land of the friction ring together with the ring land of the brake disk hub in the manner of a sandwich. Additionally, a connecting pin of the joining arrangement is arranged such that it is supported in the recesses of the support ring which correspond with the recesses of the ring lands of the brake disk hub and the friction ring.

[00011] Thereby each joining arrangement running through the recesses of the ring lands is supported by one recess of the support ring. The joining arrangements are in this manner connected to each other. The support or bearing through the support ring prevents the individual joining arrangement from inducing flexural stress onto the recesses of the ring land of the friction ring. Hence a revolving force acts onto each recess of the ring land of the friction ring from which a compression stress results in the direction of load transmission. Particularly in ceramic parts a compression stress causes much less fissures than flexural stress, which without the support ring according to the invention would act upon the ring land of the friction ring.

[00012] In one embodiment of the invention connecting pins are shrunk into the recesses of the support ring. Hence the connecting pins are perpendicular to the support ring and run through the recesses of the ring lands of the friction ring and the brake disk hub, and are screwed tight on the side of the brake disk hub. For this a thread is provided on the connecting pin on one side of the brake disk hub. A nut sits on this thread of the connecting pin and is tightened so that the support ring is pushed against the ring land of the friction ring.

[00013] In another embodiment of the invention the connecting pin features a bolt head and on its other end a thread by which it is screwed into the recess of the support ring.

[00014] The recesses of the brake disk hub can be open radially towards the outside. Thereby a radial expansion of the brake disk hub is made possible. This radial expansion of the brake disk hub can also be allowed for by a recesses in the form of a slotted hole.

[00015] The present invention is particularly advantageous for a friction ring which consists of a fiber reinforced ceramic on the basis of silicon carbide. Such friction rings exhibit a high tribological strength, are exceptionally temperature resistant and, in comparison to other ceramics, are very damage tolerant due to their fiber reinforcement.

Brief Description of the Drawings

[00016] Advantageous embodiments of the invention will be described in detail with the illustrations below. They show:

Fig. 1 a perspective view of a brake disk with a friction ring, a brake disk hub and a support ring,

Fig. 2 a cross sectional view through a brake disk with a brake disk hub, a friction ring and a support ring,

Fig. 3 a schematic illustration of the joining arrangement between friction ring and brake disk hub with a support ring,

Fig. 4 a schematic illustration of a joining arrangement between friction ring and brake disk hub with a support ring.

Detailed Description of the Invention

[00017] The brake disk 2 shown in Fig.1 includes a friction ring 4 and a brake disk hub 6. The friction ring 4 in this embodiment is comprised of a fiber reinforced silicon carbide ceramic.

[00018] The friction ring 4 as well as the brake disk hub 6 feature ring lands 8 and 10. The ring lands 8 and 10 in turn feature recesses in the friction ring 14 and in the brake disk hub 16. These recesses 14 and 16 are shown in the cross sectional view of Fig.2 and they are not visible in Fig.1.

[00019] In the brake disk 2 according to Fig. 1 there is also shown a support ring 18 which is arranged such that it encloses like a sandwich the ring land 8 of the friction ring 4 together with the ring land 10 of the brake disk hub 6. Herein "like a sandwich" means that in an axial direction first the ring land 10 of the brake disk hub 6 and then the ring land 8 of the friction ring 4 and subsequently the support ring 18 occur. The ring lands 8 and 10 and the support ring 18 feature recesses 14, 16 and 22, which are arranged such that they form a continuous bore.

[00020] A joining arrangement 12 is routed through the recesses 14, 16 and 22 (compare also Fig.2). The joining arrangement 12 comprises of a connecting pin 20 and a nut 24 (Fig.2).

[00021] The ring land 8 of the friction ring 4 exhibits slots 34 in the circumferential direction which point radially outwards and feature an end bore 36 on their outer end. The slots 34 are arranged such that they are each located equidistant between the recesses 14 in the ring land 8 of the friction ring 4. The slots 34 interrupt the ring land 8, which is advantageous in the case of

expansion caused by thermal stress. The end bores 36 serve the prevention of notch stress on the radial ends of the slots 34.

[00022] In Fig. 3 and 4 two embodiments of joining arrangements 12 are shown schematically. In the brake disk arrangement shown in Fig. 3 and 4 the disk brake is not identical with the one shown in Fig. 1 and 2. Nevertheless the same reference numbers are used for parts with similar or the same function. The embodiment in Fig. 3 features a friction ring 4 and a brake disk hub 6. Both in turn feature an ring land 8 respectively 10 which overlap each other. The ring lands 8 and 10 feature recesses 14 and 16 wherein the recesses 16 in the brake disk hub 4 are in the shape of a radial slot. This oblong recess 16 provides room for a radial expansion of the brake disk hub 6 at elevated temperatures, without inducing flexural stress in the friction ring 4.

[00023] It maybe useful to introduce a sliding block (not shown here) between the connecting pin 20 and the recess 16 which provides for a radial expansion between the joining arrangement 12 and the brake disk hub 6.

[00024] The connecting pin 20 in Fig. 3 is arranged such that it is shrunk into a recess 22 of the support ring 18. The shrinking is usually done by a heat treatment of the support ring 18, in which the cold connecting pin 20 is inserted, wherein said connecting pin 20 constitutes a frictional connection with the support ring 18 after it has cooled down.

[00025] The connecting pin 20 perpendicularly extends through the recess 22 of the support ring 18. The connecting pin 20 is guided through the recesses 14 and 16 of the ring lands 8 and 10. On the

end opposing the support ring 18 the connecting pin 20 features a thread 26. A nut 24 sits on this thread 26.

[00026] The nut 24 is tightened against the ring land 10 of the brake disk hub 6 whereby the support ring 18 is compressed against the ring land 8 of the friction ring 4. The frictional connection between support ring 18 and connecting pin 20 is sufficient to counteract a screw torque and to constitute a frictional connection between the ring lands 8 and 10 and herewith between the friction ring 4 and the brake disk hub 6.

[00027] An axial clearance between the friction ring 4 and the brake disk hub 6 may be necessary to provide for thermal expansion in the axial direction. For this, a resilient element may be utilized which is not shown in Fig. 3. Such a resilient element may be constituted by a disk spring washer between friction ring 4 and nut 24. Further, it is possible to ensure a resilient effect through an (also not shown) intermediate element between the connecting pin 20 and the recess 16.

[00028] Another schematic illustration of the joining arrangement 12 is shown in Fig. 4. Here the connecting pin 20 features a bolt head 32 on one of its ends. On its other end the connecting pin 20 features a thread 28. In this embodiment the connecting pin 20 is not shrunk into the support ring 18, but it is screwed in. Other than that, the embodiment shown in Fig. 4 is equivalent to Fig. 3.

[00029] Fig. 1 through 4 show preferred embodiments, in which the joining arrangements are constituted by a screw or threaded connection. Fundamentally the advantageous effects of the support ring can be achieved through other joining arrangements. Therein

e.g. a riveted connection or a combination of riveted connection and frictional connection, such as shrink fitting, can be utilized. It is also conceivable to use material connecting or adhesive bonds, e.g. welding or brazing.